

INFLUENCE OF TURBULENT Pr NUMBER ON FRICTION AND HEAT TRANSFER  
AT A PLATE IMMERSSED IN A TURBULENT GAS STREAM

I. P. Ginzburg and I. V. Korneva

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A solution is offered to the problem of determining the friction and heat transfer coefficients for a plate immersed in a turbulent gas stream, using the approximate dependence of heat content on velocity given in reference [1]. The influence of  $Pr_T$  on friction and heat transfer is evaluated.

The following relation, between heat content and velocity was established in reference [1] for the case of zero-gradient flow with arbitrary  $Pr_L$  and  $Pr_T$ :

$$\bar{h} = \bar{h}_w \div \left( \frac{\partial \bar{h}}{\partial \varphi} \right)_w S(\varphi) - \bar{u}^2 R(\varphi), \quad (1)$$

where

$$S(\varphi) = \int_0^{\varphi} \exp \left\{ - \int_0^{\varphi} \frac{Pr}{\omega} \frac{\partial \omega (1/Pr - 1)}{\partial \varphi} d\varphi \right\} d\varphi,$$

$$R(\varphi) = 2 \int_0^{\varphi} \exp \left( - \int_0^{\varphi} \frac{Pr}{\omega} \frac{\partial \omega (1/Pr - 1)}{\partial \varphi} d\varphi \right) \left\{ \int_0^{\varphi} Pr \left[ \exp \int_0^{\varphi} \frac{Pr}{\omega} \times \right. \right. \\ \left. \left. \times \frac{\partial \omega (1/Pr - 1)}{\partial \varphi} d\varphi \right] d\varphi \right\} d\varphi_1,$$

$$\omega = \tau/\tau_w, \quad \varphi = v_x/u, \quad \bar{h} = h/H_0, \quad \bar{u}^2 = u^2/2H_0.$$

The present paper makes use of this relation and the principles of the semi-empirical theory of turbulence to evaluate the influence of the  $Pr_T$  number on friction and heat transfer at a plate.

Approximate Form of  $R(\varphi)$  and  $S(\varphi)$  for a Two-Layer System

We shall conventionally divide the complete boundary layer into two zones: a laminar sublayer and a turbulent layer. The dimensionless velocity in these layers will vary, respectively, from 0 to  $\varphi_L$  in the laminar sublayer and from  $\varphi_L$  to  $l$  in the turbulent layer. The  $Pr$  number will take a value equal to  $Pr_L$  in the laminar sublayer and a value equal to  $Pr_T$  in the turbulent layer. We then have

$$S(\varphi) = S(\varphi_L) \div \frac{1}{Pr_L} \int_{\varphi_L}^{\varphi} Pr_T \left( \exp - \int_0^{\varphi} \frac{1 - Pr}{\omega} \frac{\partial \omega}{\partial \varphi} d\varphi \right) d\varphi_1;$$

$$R(\varphi) = R(\varphi_L) \div 2 \int_{\varphi_L}^{\varphi} Pr_T \left( \exp - \int_0^{\varphi} \frac{1 - Pr}{\omega} \frac{\partial \omega}{\partial \varphi} d\varphi \right) \times \\ \times \left[ \int_0^{\varphi} \left( \exp \int_0^{\varphi} \frac{1 - Pr}{\omega} \frac{\partial \omega}{\partial \varphi} d\varphi \right) d\varphi \right] d\varphi_1.$$

We represent  $\omega(\varphi)$  approximately in the form of a third-degree polynomial:

$$\omega = 1 - \varphi^3.$$

Assuming that  $Pr_T$  and  $Pr_L$  are constants, we calculate  $S(\varphi_L)$  and  $R(\varphi_L)$ . Making a numerical approximation to the re-

sults obtained, we have

$$S(\varphi_L) = \varphi_L, \quad R(\varphi_L) \approx Pr_L \varphi_L^2.$$

The integrals  $S(\varphi)$  and  $R(\varphi)$ , with account for the form of  $S(\varphi_L)$  and  $R(\varphi_L)$ , become

$$\begin{aligned} S(\varphi) &= \varphi_L \left[ 1 - \frac{Pr_\tau}{Pr_L} \omega_L^{Pr_L - Pr_\tau} \right] + \frac{Pr_\tau}{Pr_L} \omega_L^{Pr_L - Pr_\tau} I_1, \\ R(\varphi) &= \varphi_L^2 (Pr_L - Pr_\tau) + 2Pr_\tau I_2, \end{aligned} \quad (2)$$

where

$$I_1 = \int_0^{\infty} \omega^{Pr_\tau - 1} d\varphi, \quad I_2 = \int_0^{\infty} \omega^{Pr_\tau - 1} \left( \int_0^{\infty} \omega^{1 - Pr_\tau} d\varphi \right) d\varphi.$$

Calculation of  $I_1$  and  $I_2$  for  $Pr_\tau$  varying from 0.5 to 2 shows that  $I_1$  and  $I_2$  may be put in the approximate form

$$I_1 = a_1 \varphi, \quad I_2 = a_2 \varphi^2, \quad (3)$$

where

$$a_1 = 1.214 - 0.214 Pr_\tau, \quad a_2 = 0.65 - 0.15 Pr_\tau.$$

As regards  $I_1(1)$  and  $I_2(1)$ ,  $I_1(1)$  may be accurately expressed in terms of gamma functions, and  $I_2(1)$  approximately, as follows:

$$\begin{aligned} I_1(1) &= \frac{1}{3} \frac{\Gamma(1/3) \Gamma(Pr_\tau)}{\Gamma(Pr_\tau + 1/3)}, \\ I_2(1) &\approx \frac{1}{3} \frac{\Gamma(2/3) \Gamma(Pr_\tau)}{\Gamma(Pr_\tau + 2/3)} - \frac{1 - Pr_\tau}{12} \frac{\Gamma(5/3) \Gamma(Pr_\tau)}{\Gamma(5/3 + Pr_\tau)}. \end{aligned}$$

Determining  $(\partial h / \partial \varphi)_w$  from the boundary conditions  $h = h_0$  when  $\varphi = 0$ , we obtain

$$\left( \frac{\partial \bar{h}}{\partial \varphi} \right)_w = \frac{\bar{h}_0 + \bar{u}^2 R(1) - \bar{h}_w}{S(1)}, \quad (4)$$

where

$$\begin{aligned} S(1) &= \alpha \varphi_L + \beta I_1(1), \\ R(1) &= \gamma \varphi_L^2 + 2Pr_\tau I_2(1), \\ \alpha &= 1 - \frac{Pr_\tau}{Pr_L} \omega_L^{Pr_L - Pr_\tau}; \quad \beta = \frac{Pr_\tau}{Pr_L} \omega_L^{Pr_L - Pr_\tau}, \\ \gamma &= Pr_L - Pr_\tau. \end{aligned}$$

Substituting (2) into (1) and taking (3) into account, we can approximate to the dependence of  $h$  on  $\varphi$  in the turbulent layer by means of a second-degree polynomial in  $\varphi$ :

$$\bar{h} = A \varphi^2 + B \varphi + C, \quad (5)$$

where

$$\begin{aligned} A &= -2Pr_\tau a_2 \bar{u}^2, \\ B &= (1 - \bar{h}_w + [R(1) - 1] \bar{u}^2) \beta a_1 / S(1), \\ C &= \bar{h}_w + (1 - \bar{h}_w + [R(1) - 1] \bar{u}^2) \alpha \varphi_L / S(1) - \bar{u}^2 \gamma \varphi_L. \end{aligned}$$

In the laminar sublayer

$$\begin{aligned} S(\varphi) &\approx \varphi, \quad R(\varphi) = Pr_L \varphi^2, \\ \bar{h} &= \bar{h}_w + \left( \frac{\partial \bar{h}}{\partial \varphi} \right)_w \varphi - \bar{u}^2 Pr_L \varphi^2 = \bar{h}_w + B' \varphi + A' \varphi^2. \end{aligned} \quad (6)$$

## Derivation of the Velocity Profile

We shall determine the velocity distribution law over the boundary layer.

In accordance with the basic assumptions of the semi-empirical theory of turbulence, the friction stress is given by:

$$\begin{aligned} y \geq \delta_L, \quad \tau &= \rho k^2 y^2 u^2 (\partial \varphi / \partial y)^2, \\ y \leq \delta_L, \quad \tau &= \mu u \partial \varphi / \partial y. \end{aligned}$$

In accordance with the preceding paragraph, we make the assumption

$$\tau / \tau_w = 1 - \varphi^3.$$

We shall further suppose that

$$\rho / \rho_w = h_w / h = \bar{h}_w / \bar{h}.$$

Using the expressions derived, after substituting  $\bar{h}$  from (5), for determining the dependence  $\varphi(y)$  we obtain

$$y \leq \delta_L, \quad \mu u \frac{\partial \varphi}{\partial y} = \tau_w (1 - \varphi^3); \quad (7)$$

$$y \geq \delta_L, \quad \frac{\tau_w}{\rho_w u^2} \frac{1}{k^2 y^2} = \frac{\bar{h}_w (\partial \varphi / \partial y)^2}{(1 - \varphi^3)(A \varphi^2 + B \varphi + C)}. \quad (8)$$

Integrating (8) approximately, and taking into account that  $A < 0$ , on condition that  $\varphi = 1$  when  $y = \delta$ , we obtain

$$\ln \frac{y}{\delta} = \sqrt{\bar{h}_w} k \zeta \frac{1,107}{\sqrt{-A}} \left( \arcsin \frac{-2A\varphi - B}{\sqrt{B^2 - 4AC}} - \arcsin \frac{-2A - B}{\sqrt{B^2 - 4AC}} \right). \quad (9)$$

In the laminar sublayer, assuming that  $\mu = \mu_w (h/h_w)^n$ , after substituting the value of  $h$  from (6) into (7), for determining  $\varphi(y)$  we obtain the equation

$$\left( 1 + \frac{B'}{h_w} \varphi + \frac{A'}{h_w} \varphi^2 \right) \frac{\partial \varphi}{\partial y} \approx \frac{\tau_w}{\mu_w u}, \quad (10)$$

whose solution may be written approximately as:

$$\varphi \left( 1 + \frac{nB'}{h_w} \varphi + \frac{nA'}{h_w} \varphi^2 \right) = \frac{\tau_w}{\mu_w u} y.$$

This formula is valid for  $0 \leq y \leq \delta_L$ .

## Determination of the Thickness of the Laminar Sublayer and the Flow Velocity at its Edge

The velocity derivative has a discontinuity at the edge of the laminar sublayer

$$\left( \frac{\partial v_x}{\partial y} \right)_{y=\delta_L-0} = k_1 \left( \frac{\partial v_x}{\partial y} \right)_{y=\delta_L+0}. \quad (11)$$

From the relations of the semi-empirical theory,  $\mu \frac{\partial v_x}{\partial y} = \tau$  when  $y \leq \delta_L$ ,  $\tau = \rho l^2 \left( \frac{\partial v_x}{\partial y} \right)^2$ , and when  $y \geq \delta_L$ , assuming  $l = ky$ , we find

$$\left( \frac{\partial v_x}{\partial y} \right)_{y=\delta_L-0} = \frac{\tau_L}{\mu_L}; \quad \left( \frac{\partial v_x}{\partial y} \right)_{y=\delta_L+0} = \frac{1}{k \delta_L} \sqrt{\frac{\tau_L}{\rho_L}}.$$

Substituting into (11), we obtain

$$\delta_L = \frac{k_1}{k} \frac{\mu_L}{\sqrt{\rho_L \tau_L}}. \quad (12)$$

Since  $\delta_L / \delta$  is small, we may put  $\tau_L \approx \tau_w$ . Taking into account that  $\rho / \rho_w = h_w / h$ ,  $\mu = \mu_w (h/h_w)^n$  we obtain

$$\frac{\mu_L}{\sqrt{\rho_L}} = \frac{\mu_w}{\sqrt{\rho_w}} \left[ 1 + \left( n + \frac{1}{2} \right) \frac{B'}{h_w} \varphi_L + \left( n + \frac{1}{2} \right) \frac{A'}{h_w} \varphi_L^2 \right].$$

Substituting  $\mu_L/\sqrt{\rho_L}$  into (12), we obtain

$$\delta_L = \frac{k_1}{k} \frac{\mu_w}{\rho_w \sqrt{\tau_w/\rho_w}} \left[ 1 + \left( n + \frac{1}{2} \right) \frac{B'}{\bar{h}_w} \varphi_L + \left( n + \frac{1}{2} \right) \frac{A'}{\bar{h}_w} \varphi_L^2 \right]. \quad (13)$$

To determine  $\varphi_L$ , we put  $y = \delta_L$ ,  $\varphi = \varphi_L$  in (10), and obtain

$$\begin{aligned} & \varphi_L \left( 1 + \frac{n}{2} \frac{B'}{\bar{h}_w} \varphi_L + \frac{n}{3} \frac{A'}{\bar{h}_w} \varphi_L^2 \right) = \\ & = \frac{k_1}{k} \sqrt{\frac{\tau_w}{\rho_w}} \frac{1}{u} \left[ 1 + \left( n + \frac{1}{2} \right) \frac{B}{\bar{h}_w} \varphi_L + \left( n + \frac{1}{2} \right) \frac{A'}{\bar{h}_w} \varphi_L^2 \right]. \end{aligned} \quad (14)$$

This equation may be solved by the method of successive approximations. We take as the first approximation

$$\varphi_L = \frac{k_1}{k} \frac{v_*}{u} = \frac{k_1}{k \zeta}, \quad \delta_L = \frac{k_1}{k} \frac{v_w \zeta}{u},$$

where

$$\zeta = \frac{u}{\sqrt{\tau_w/\rho_w}} = \frac{u}{v_*}.$$

#### Deviation of the Relation Between $\zeta$ and $\delta$

Putting  $y = \delta_L$ ,  $\varphi = \varphi_L$  in (9), we obtain the following equation relating  $\zeta$  and  $\delta$ :

$$\ln \frac{\delta_L}{\delta} = \frac{1.107}{\sqrt{-A}} k \zeta \sqrt{\bar{h}_w} \left( \arcsin \frac{-2A\varphi_L - B}{\sqrt{B^2 - 4AC}} - \arcsin \frac{-2A - B}{\sqrt{B^2 - 4AC}} \right). \quad (15)$$

Replacing  $\delta_L$  and  $\varphi_L$  by their values in the first approximation, taking into account that  $1/k\zeta$  is small, expanding the terms in brackets in series, and retaining terms containing  $1/k\zeta$  in the first degree, we obtain

$$u \delta/v_w = Dk\zeta \exp(Ck\zeta/u).$$

Here

$$\begin{aligned} C &= 1.107 \sqrt{\frac{\bar{h}_w}{2\text{Pr}_\tau a_2}} \left\{ \arcsin \frac{1 - \bar{h}_w + \bar{l}\bar{u}^2}{\varepsilon} - \arcsin \frac{1 - \bar{h}_w + \bar{f}\bar{u}^2}{\varepsilon} \right\}; \\ l &= 2\text{Pr}_\tau I_2(1) - 1; \quad f = l - 4\text{Pr}_\tau a_2 I_1(1) a_1; \\ \varepsilon^2 &= (1 - \bar{h}_w + \bar{l}\bar{u}^2)^2 + 8\text{Pr}_\tau a_2 \bar{u}^2 \bar{h}_w I_1^2(1) a_1^2; \\ D &= \frac{k_1}{k^2} \exp \left\{ -1.107 k_1 \left( 1 - \frac{\text{Pr}_\tau - \text{Pr}_\pi}{\varepsilon^2 a_1 \text{Pr}_\tau} \times \right. \right. \\ & \times (1 - \bar{h}_w + \bar{l}\bar{u}^2) \left[ 1 + \bar{h}_w + \bar{l}\bar{u}^2 - 2 \left\{ \frac{\bar{h}_w}{2} \left( 1 - \frac{a_1}{I_1} \right) + \frac{1}{2} \left( 1 + \frac{a_1}{I_1} \right) - \right. \right. \\ & \left. \left. - \bar{u}^2 \left[ 2\text{Pr}_\tau a_2 \frac{I_1}{a_1} - \frac{1}{2} \left( \frac{a_1}{I_1} + 1 \right) l \right] \right\} \frac{\sqrt{\bar{h}_w}}{\sqrt{1 - \bar{u}^2}} \right] \right\}. \end{aligned}$$

In order to obtain the friction drag of the plate, we must find a second equation relating the friction and the boundary layer thickness.

We obtain this equation by using the integral relation expressing the momentum law.

#### Determination of the Ratio $\delta^{**}/\delta$

By definition

$$\frac{\delta^{**}}{\delta} = \int_0^1 \frac{\rho}{\rho_0} \varphi (1 - \varphi) d \frac{y}{\delta} = \frac{\rho_w}{\rho_0} I.$$

Since  $\delta_L/\delta \ll 1$ , in determining  $\delta^{**}/\delta$  we shall take values of  $\varphi$  and  $\rho/\rho_w$ , for the turbulent regime. To calculate  $\delta^{**}/\delta$ , we integrate successively by parts and put the result in the form of a series in  $1/k\zeta$ :

$$I = \int_0^1 \frac{\rho}{\rho_w} (\varphi - \varphi^2) d \frac{y}{\delta} = \frac{\sqrt{\bar{h}_w}}{1.107k\zeta} \frac{1}{\sqrt{1-\bar{u}^2}} + \frac{F_1}{k\zeta^2} + \frac{F_2}{k\zeta^3} + \dots$$

Then, taking account of the expression for  $\rho_w/\rho_0$ , we obtain

$$\frac{\delta^{**}}{\delta} = \sqrt{\frac{1-\bar{u}^2}{\bar{h}_w}} \frac{1}{1.107k\zeta} + \dots$$

### Determination of the Friction Law

We use the integral relation expressing the momentum law

$$\frac{d}{dx} \rho_0 u^2 \delta^{**} + \rho_0 u \frac{du}{dx} \delta^{**} = \tau_w.$$

For the plate  $u = \text{const}$  and  $\rho_0 = \text{const}$ , and therefore  $\rho_w = \text{const}$  also; making the assumption that  $T_w = \text{const}$ , we may write this equation as

$$\frac{d}{dx} \left( \frac{u\delta}{\nu_w} I \right) = \frac{u}{\nu_w} \frac{\tau_w}{\rho_w u^2} = \frac{1}{\zeta^2} \frac{u}{\nu_w}.$$

Replacing  $I$  and  $u\delta/\nu_w$  by their values, evaluating the last integral approximately, and taking into account that  $1/k\zeta$  is small, we obtain

$$\frac{ux}{\nu_w} = D\zeta^2 \exp \frac{C}{u} k\zeta \sqrt{\frac{\bar{h}_w}{1-\bar{u}^2}} \left( \frac{1}{1.107} + \frac{F_1}{k\zeta} + \frac{F_2}{k\zeta^2} + \dots \right). \quad (16)$$

The local friction coefficient is

$$c_f = \frac{2\tau_w}{\rho_0 u^2} = \frac{2}{\zeta^2} \frac{1-\bar{u}^2}{\bar{h}_w}.$$

The friction coefficient for a plate of length  $l$  will be

$$C_f = \int_0^l \frac{c_f}{l} dx = \frac{2\delta_{x=l}^{**}}{l} = \frac{\nu_w}{u} \left( \frac{u\delta}{\nu_w} I \right)_{x=l} \frac{2}{l} \frac{1-\bar{u}^2}{\bar{h}_w}.$$

Substituting for  $u\delta/\nu_w$  and  $I$ , we obtain

$$C_f = \frac{2}{1.107} \sqrt{\frac{1-\bar{u}^2}{\bar{h}_w}} \frac{\nu_w}{ul} D \exp \frac{C}{u} k\zeta_l, \quad (17)$$

where we determine  $\zeta_l$  from (16), putting  $x = l$ . For an incompressible fluid  $C_f$  is obtained for  $\bar{u} = 0$ ,  $\bar{h}_w = 1$ ,  $\nu_w = \nu_0$ .

### Interpolation Formula for $C_f$

The formulas obtained for  $c_f$  and  $C_f$  may be simplified within a certain range of variation of  $\zeta$ . When  $\zeta$  varies in the limits 8 to 20, as shown in [2], the function  $\ln(\zeta^2 \exp \zeta)$  may be represented approximately as the straight line:

$$\ln(\zeta^2 \exp \zeta) = n_1 + n_2 \zeta,$$

where  $n_1 = 2.9$ ;  $n_2 = 1.16$  for the range of variation of  $\zeta$  indicated. In this case

$$\left[ \exp \frac{C}{u} k\zeta \right] k^2 \zeta^2 = \frac{u^2}{C^2} \exp(n_1 + n_2 \bar{\zeta}), \quad \bar{\zeta} = Ck\zeta/\bar{u}.$$

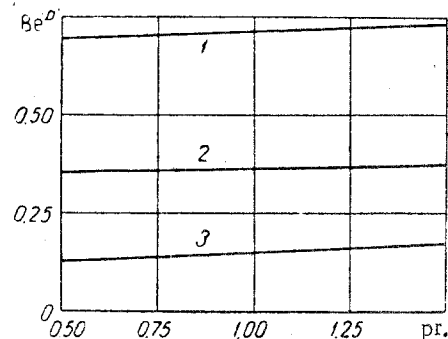


Fig. 1. Dependence of  $Be^D$  on  $Pr_T$  for  $h_w = 1$  and  $Pr_L = 1$ ; 1-M = 2; 2-5; 3-10.

Formula (16) may be written in the form

$$\frac{ux}{\nu_w} = \frac{D\bar{u}^2}{k^2 C^2} \exp(n_1 + n_2 \bar{\zeta}) \left( \sqrt{\frac{\bar{h}_w}{1 - \bar{u}^2}} \frac{1}{1.107} + \dots \right). \quad (18)$$

Putting  $x = l$  and neglecting the terms  $\frac{F_1}{k\zeta} + \frac{F_2}{(k\zeta)^2}$  in comparison with the first, we obtain

$$n_1 + n_2 \bar{\zeta}_l = \ln \left( \frac{k^2 C^2}{D\bar{u}^2} \frac{ul}{\nu_w} \sqrt{\frac{1 - \bar{u}^2}{\bar{h}_w}} 1.107 \right).$$

Substituting  $\bar{\zeta}_l$  from this formula into (17), we obtain

$$C_f = (1.107)^{\frac{3-n_2}{n_2}} \left( \frac{ul}{\nu_0} \right)^{\frac{1-n_2}{n_2}} D^{\frac{n_2-1}{n_2}} \left( \frac{c}{u} \right)^{\frac{2}{n_2}} (\bar{h}_w)^{\frac{n_2-1}{2n_2}} (1 - \bar{u}^2)^{\frac{3-n_2}{2n_2}} \times \\ \times \left( \frac{\mu_w}{\mu_0} \right)^{\frac{n_2-1}{n_2}} 2k^{\frac{2}{n_2}} \exp\left(-\frac{n_1}{n_2}\right) (2Pr_T a_2)^{-\frac{1}{n_2}}.$$

Substituting the value of C and D and taking into account that  $n_1 = 2.9$ ;  $n_2 = 1.16$ ;  $k = 0.39$ ;  $k_1/k = 11$ , we obtain

$$C_f = 0.027 Re^{-0.139} B(\bar{u}, \bar{h}_w) \exp D', \\ B = (\bar{h}_w)^{0.0695} (1 - \bar{u}^2)^{0.79} \left( \frac{\mu_w}{\mu_0} \right)^{0.139} (2Pr_T a_2)^{-0.861} \times \\ \times \left[ \frac{1}{\bar{u}} \left( \arcsin \frac{\bar{h}_w - 1 - \bar{f}\bar{u}^2}{\varepsilon} - \arcsin \frac{\bar{h}_w - 1 - \bar{l}\bar{u}^2}{\varepsilon} \right) \right]^{1.722}, \\ D' = 0.139 \cdot 1.107 k_1 \frac{Pr_T - Pr_n}{\varepsilon^2 Pr_T a_1} (1 - \bar{h}_w + \bar{l}\bar{u}^2) \left[ 1 + \bar{h}_w + \bar{l}\bar{u}^2 - 2 \left( \frac{\bar{h}_w}{2} \left( 1 - \frac{a_1}{I_1} \right) + \right. \right. \\ \left. \left. + \frac{1}{2} \left( 1 + \frac{a_1}{I_1} \right) - \bar{u}^2 \left( 2Pr_T a_2 \frac{I_1}{a_1} - \frac{1}{2} \left( \frac{a_1}{I_1} + 1 \right) l \right) \right] \sqrt{\frac{\bar{h}_w}{1 - \bar{u}^2}} \right].$$

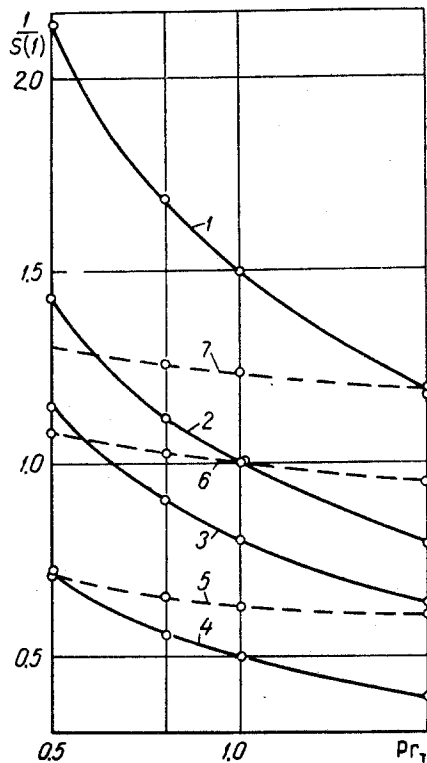


Fig. 2. Dependence of  $1/S(1)$  on  $Pr_T$  and  $Pr_L$  for  $\varphi_L = 0(1 - Pr_L = 1.5; 2 - 1.0; 3 - 0.8; 4 - 0.5)$  and for  $\varphi_T = 0.5(5 - Pr_L = 0.5; 6 - 1; 7 - 1.5)$ .

The factor  $B(\bar{u}, \bar{h}_w) \exp D'$  characterizes the influence of the variable heat content and compressibility on the friction drag of the plate. The factor  $\exp D'$  characterizes the influence of  $Pr_L$  on the value of  $C_f$ . When  $Pr_T = 1$ , we obtain the formula given in [2] for  $C_f$ .

To evaluate the influence of  $Pr_T$  on the friction drag, we give the dependence of  $Be^{D'}$  on  $Pr_T$  for  $\bar{h}_w = 1$  and for various  $M$  (Fig. 1). It may be seen from the graph that the influence of  $Pr_T$  on  $C_f$  is insignificant. The influence of  $Pr_T$  on  $q_w$  is more substantial, as may be seen from the expression for  $S(1)$  and  $R(1)$ .

#### Influence of $Pr_T$ on Heat Transfer

Using the expression for  $h(\varphi)$ , derived at the beginning of this article, we obtain that

$$q_w = \frac{\lambda_w}{C_{p_w}} \frac{\partial h}{\partial y} \Big|_{y=0} = \frac{\lambda_w}{C_{p_w}} \left( \frac{\partial h}{\partial \varphi} \right)_w \frac{1}{u} \frac{\partial v_x}{\partial y} \Big|_{y=0}.$$

Substituting the value of  $\left( \frac{\partial h}{\partial \varphi} \right)_w$  from (4), and taking into consideration that  $\frac{\partial v_x}{\partial y} \Big|_{y=0} = \frac{\tau_w}{\mu_w}$ , we have

$$q_w = \frac{1}{Pr(0)} \frac{\tau_w}{u} \frac{h_r - h_w}{S(1)},$$

where

$$h_r = h_0 + R(1) u^2/2.$$

For the dimensionless heat transfer coefficient we obtain the expression

$$St = \frac{q_w}{(h_r - h_w) \rho_0 u} = \frac{c_f}{2Pr(0)} \frac{1}{S(1)}.$$

Figure 2 shows the dependence of  $1/S(1)$  as a function of  $Pr_L$  and  $Pr_T$  for two values of  $\varphi_L$ . It follows from the graphs in this figure that  $Pr_T$  has an appreciable influence on the value of the  $St$  number.

#### NOTATION

$v_x, v_y$ —velocity components along the coordinate axes;  $p, \rho, \mu, \lambda$ —pressure, viscosity and thermal conductivity respectively;  $h$ —enthalpy of unit mass of gas;  $H = h + v_x^2/2$ —total heat content of unit mass of gas;  $\tau$  and  $q_y$ —components of friction stress tensor and heat flux vector;  $R$ —gas constant;  $M$ —molecular weight;  $Pr$ —Prandtl number;  $u, \rho_0, h_0, H_0 = h + u^2/2$ —values characterizing the external flow;  $\rho_w$ —density at wall;  $\delta_L$ —thickness of laminar sublayer;  $\varphi_L$ —velocity at edge of laminar sublayer.

#### REFERENCES

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Zhdanov State University, Leningrad